

Simulation and Optimization of Hydraulics Applied in Oil Well Drilling

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Abstract. The main objective of hydraulics applied in oil well drilling is to ensure cleanliness at the bottom of the well, this consists of keeping in continuous movement the solids or cuttings of the formation, transporting the largest amount of them from the bottom and the annular space to the surface, thus minimizing the existence of unwanted events such as sticking, entrapment, among others. Each oil well has specific geological, geophysical and geomechanical characteristics that must be taken into account for its design, based on technical and scientific considerations that allow the establishment of its mechanical state and its exploitation engineering studies. The rheological behavior of a fluid in these wells cannot be generalized, even if they are separated by a few hundred meters. This paper describes the development of a graphical computational model for the fast and efficient calculation of the various hydraulic aspects associated with onshore oil well drilling. The model is arranged in three stages: First, the position of the drill pipes in the drillstring is evaluated and their pull margin is determined. Then, the hydraulic calculation procedures of national and international companies are

developed. Finally, rheological models (Bingham plastic and exponential law) are used for fluid behavior. The model was developed with Microsoft Visual Basic.

Keywords. Drilling hydraulics, rheology, bit pressure, annular pressure, equivalent circulating density.

1 Introduction

The Microsoft Excel calculation memory is one of the basic and universally used tools in engineering and has been enhanced with the inclusion of the Visual Basic language as a programming and task automation tool.

Since its first versions, Visual Basic has allowed the development of several engineering applications due to its ease of learning, simple and immediate implementation, and its versatility that allows the development of programming instructions according to specific needs of the oilwell drilling industry, such as gas correlations,

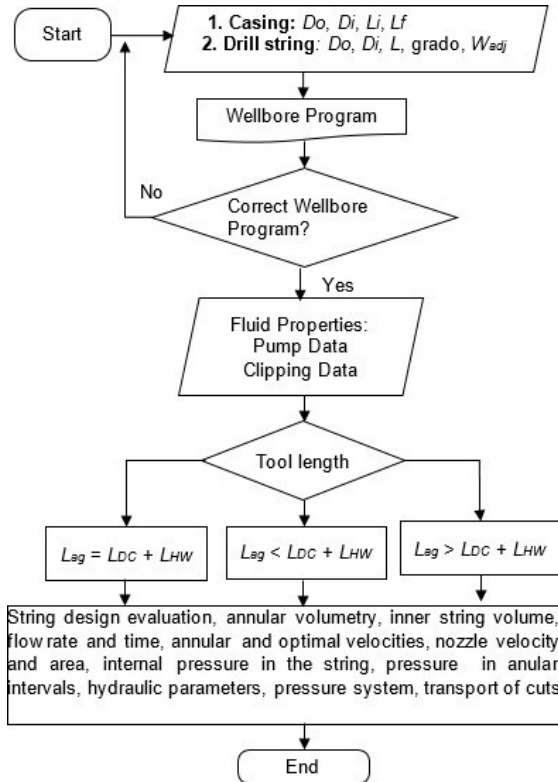


Fig. 1. Drilling fluids Hydraulics I diagram

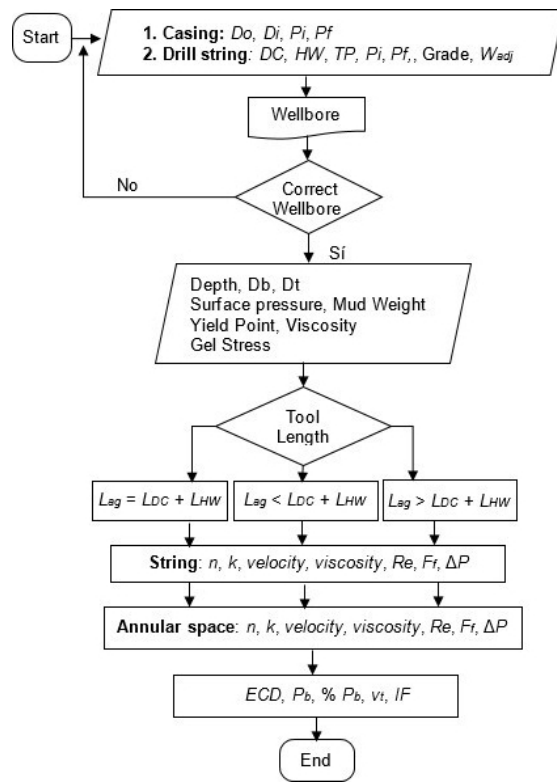


Fig. 2. Drilling fluids Hydraulics II diagram

volumetric reserves calculations, simple log analysis, water pattern analysis, among others [17].

The oil industry has undergone a transformation thanks to the advances in computer industry and the introduction of the Internet, which, among other applications, have enabled log acquisition and analysis, reservoir simulation, well testing, production and reserve data analysis, as well as the filling of regulatory reports [7].

Hydraulics plays an important role in many oilfield operations, including drilling, cementing, completion, fracturing, acidizing, production, and reconditioning.

However, in the case of drilling, the role of hydraulics becomes vital, as optimized hydraulics can minimize cost and, conversely, miscalculations can cause problems such as fluid loss, or can even lead to the loss of the well [10].

Drilling is one of the various activities in the oil industry whose main objective is to make a physical connection between the reservoir and the surface.

One of the first computer applications developed was the simulation of a circulation system during drilling; this algorithm had the ability to simulate the washing of the drill string, losing fluid and fracturing the formation [9].

Any hydraulic design developed in well drilling is based on minimizing drilling stresses and reducing associated costs [8]. However, hydraulic modeling is an integral part of the various drilling operations for the realization in an efficient well, the API (American Petroleum Institute) standards consider the Bingham plastic and exponential law models because they provide a simple way to estimate the necessary parameters for efficient drilling of conventional wells [18].

Fig. 3. Software cover page with user data and well location

The importance of optimizing drilling fluid properties lies in ensuring drilling operations and process safety [14]. In consideration, the evaluation and prediction of well cleanliness presents a challenge in the drilling industry, since good well cleanliness means a high penetration rate and fewer drilling problems [1].

Additionally, it has been observed that drilling fluids with similar properties according to the API standard can have significantly different behavior with respect to cuttings transport efficiency, resulting this in a research topic in recent years [15].

Still, nowadays these properties are manually measured and optimized by engineers with different skills and experiences that at a certain moment could lead to non-optimal characteristics of the drilling fluid, i.e., cause a deterioration of its functionalities.

To minimize these events, the use of software would allow the study of various scenarios that could be supported by the experiences of design and operating engineers, especially when wellbore cleanliness and stability conditions change during drilling.

One of the areas of opportunity in the study plans and programs of technical and university bachelor's degrees related to petroleum engineering is the lack of simulation software, that allows the development of design and analysis skills for well drilling and fluid exploitation processes in accordance with relevant technical standards and engineering practice.

On the contrary, several national and international oil sector companies have developed their own computational models that are restricted for use by their personnel or are commercially available at prohibitive license costs that are unaffordable for higher education institutions.

In addition, well drilling fluid hydraulics software represents, within the teaching-learning process, a considerable reduction of time compared to manual calculations, thus giving the opportunity to perform a greater number of design simulations and the consequent analysis of alternatives in oil well drilling.

The objective of the present work was to develop a computational model in the Microsoft Visual Basic environment whose purpose is to simulate and optimize the hydraulic processes related to onshore oil well drilling.

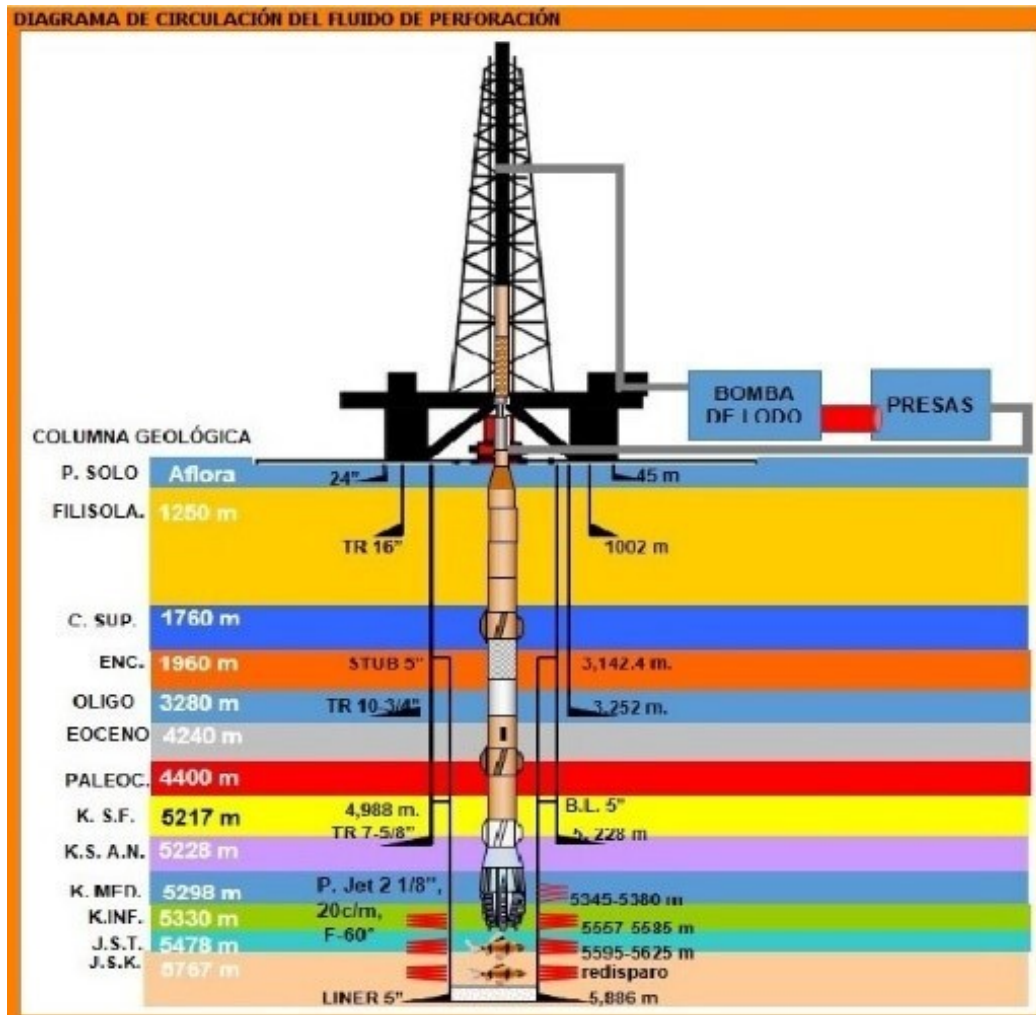


Fig. 4. Drilling fluid circulation diagram

This model considers two rheological models (Bingham plastic and exponential law) to comprehensively simulate variable rheological behavior, as reported in technical literature [3]; that is, a drilling fluid rheology could match a model behavior at the surface and act as another fluid downhole.

Some studies have revealed that mud rheology, density, transport velocity, tubing rotation and well depth are the controlling factors that influence wellbore cleanup [5]. The model presented here meets a series of desirable technical considerations, such as: i) ease of use, ii)

simulation options in line with real operational activities, iii) accurate and validated calculations, iv) unnecessary periodic software updates, v) use of the software without the need for technical support, and vi) low memory space requirements and compatibility with Excel 2010 and later.

Likewise, this software brings together valuable academic aspects since it allows: i) technical training for inexperienced professionals, ii) problem-based learning for technical and university degree students, and iii) the use of two well hydraulics methodologies and two rheological models.

Table 1. Initial information required by the program (national methodology)

Element			Dimensions and Properties			
Casing Pipe			D_i (in)	Depth L (m)		
Driven			29	150		
Superficial			23	600		
Intermediate			17	1,200		
Exploitation			11	2,100		
Borehole			9	2,100 – 2,500		
Drillstring			D_i (in)	D_o (in)	L (m)	W_{adj} (kg/m)
Drill collar			7	2.25	2,500 – 3,000	136
Heavy weight			5	3.00	2,300 – 2,200	74.5
Drill pipe						
Section	Degree	R (kg)	D_i (in)	D_o (in)	L (m)	W_{adj} (kg/m)
Tp1	E	127,181	4	3.00	2,200 – 900	31.12
Tp2	X	161,096	4	3.00	900 – 300	31.94
Tp3	G	178,054	4	3.00	300	32.66
Pump (triplex type)			D_p (in)	L_p (in)	η (%)	Q (spm)
			6.5	12	90	105
Fluid Properties			ρ (kg/m ³)	σ_y (lb/ft ²)	μ_p (cP)	μ_a (cP)
			1560	0.1	24	22.7
Nozzels			N_t	D_t (in)		
			3	16/32		
Cuttings			Lenght (cm)	Width (cm)	Thickness (cm)	
Dimensions			4	3	3	
Density (kg/m ³)			2,700			
Other Dimensions						
Steel density (kg/m ³)			7,850			
Total depth (m)			2,500			
Surface pressure (kPa)			345 (50 psi)			
Bit diameter (in)			9 in			

2 Methodology

Drilling engineering is very complex, so much so that the design of a program contains so many important variables of study ranging from the physicochemical composition of a fluid, casing selection and setting, drill string design, bit types and surface connections, to name a few.

Four sections were developed for this program: I) Cover, II) Hydraulics I, III) Hydraulics II, and IV) Rheological Models.

2.1 Cover

Cover section is integrated by basic information of the well, such as name, number, classification, objectives, and geographic coordinates of the well.

Método 1/Method 1 | Volumetry and Hydraulics | Método 2/Method 2 | Volumetry and Hydraulics | Rheological Models

Casing Pipe Information				Basic Information			
Casing Name	Diámetro Interior (") Inside Diameter	Prof. Inicial (m) Starting Depth	Profundidad (m) Depth	Depth (m)	Density (gr/cm ³)	Fluid Type	Yield Point (lb/100ft ²)
Casing Driver	29	0	150	2500	1.56	Base agua	10
Surface Casing	23	0	600				24
Intermediate Casing	17	0	1200				7.85
Exploitation Casing	11	0	2100				50
Bare Hole	9	2100	2500				22.71

Información de Sarta de Perforación/Drill String Information							
Herramienta Tool	Diá Exterior (") External Diameter	Diá Interior (") Inside Diameter	Prof. Inicial (m) Starting Depth	Prof. Final (m) Final Depth	Grado (X,E,G,S) Grade	P Ajustado(kg/m) Adjusted Weight	R Tensión(kg) Tensile Strength
Drill Collar	7	2.25	2500	2300		136	
Heavy Weight	5	3	2300	2200		74.5	
Drill Pipe 1	4	3	2200	900	E	31.12	127181
Drill Pipe 2	4	3	900	300	X	31.94	161096
Drill Pipe 3	4	3	300	0	G	32.66	178054

Mud Bomb		Barrenas/Bit		Cut Data	
Piston Diameter (in)	6.5	Bit (in)	9	Long	4
Stroke Length (in)	12	Nozzles	3	Width	3
Volumetric Efficiency (%)	0.90	RPM Bit		Thickness	3
Operating Flow (epm)	105	Weight on Bit (Ton)		Density(gr/cm ³)	2.7
Pump Pressure (psi)		Nozzle Diameter	16		
		(?/32")	16		

Wellbore Program
Wellbore Program Delete
Atras/Behind
Siguiete/Following

Estado Mecánico de Hidráulica/Hydraulic Wellbore Program

Fig. 5. Data entry and mechanical state section

Método 1/Method 1 | Volumetry and Hydraulics | Método 2/Method 2 | Volumetry and Hydraulics | Rheological Models

Diseño de Sarta/String Design		Drillstring Inner Volume		Nozzle Area and Speed		Pressure in annular intervals	
Factor de Flotación/Float factor	0.8013	V Interior TP3(R)/Inner volume	1368.09000	Área Tob1(pulg ²)/Nozzle Area 1	0.19635	Annular Pressure (AG-DC) (lb/pulg ²)	45.95435
Peso Flotado TP3 (kg)/Float weight	7851.13740	V Interior TP2(R)/Inner volume	2736.18000	Área Tob2(pulg ²)/Nozzle Area 2	0.19635	Annular Pressure (AG-HW) (lb/pulg ²)	3.66228
Peso Flotado TP2(kg)/Float weight	15356.11320	V Interior TP1(R)/Inner volume	5928.39000	Área Tob3(pulg ²)/Nozzle Area 3	0.19635	Annular Pressure (TR-HW) (lb/pulg ²)	
Peso Flotado TP1(kg)/Float weight	32417	V Interior HW(R)/Inner volume	456.03000	Área Total (pulg ²)/Total Area	0.58905	Annular Pressure (AG-TP1) (lb/pulg ²)	2.14584
Peso Flotado HW (kg)/Float weight	5969.88500	V Interior DC(R)/Inner volume	513.03375	Vel Tobera(ft/seg)/Nozzle speed	285.58901	Annular Pressure (TR-TP1) (lb/pulg ²)	7.23246
Peso Flotado DC (kg)/Float weight	21795.36000	V Interior Total (R)/Total Inner	11001.72375	CPbna(lb/pulg ²)/Bit pressure	824.70840	Annular Pressure (TR-TP2) (lb/pulg ²)	3.61623
Peso Total Sarta(kg)/Float weight	83390.48967			CPbna(kg/cm ²)/Bit pressure	57.99637	Annular Pressure (TR-TP3) (lb/pulg ²)	1.80811
						Total Annular Pressure (lb/pulg ²)	64.41928
						Total Annular Pressure (kg/cm ²)	4.53019

Expenses and Circulation Times		Internal Pressure in Drillstring		Scrap Transport		
Sarta1: TP1, TP2, TP3/String	43790.51033	Bomba Triplex	1850.45946	VARec (m/min)	10.23078	TIEMPO
	110122.51033	T Atraso (hras)/Delay Time	65.44391	VRasc (AG-DC) (m/min)	103.85848	1.92570
	142436.62353	T Desplamiento (hras)/Travel Time	71.38931	VRasc (AG-HW) (m/min)	54.96308	1.81940
Sarta2: TP2, TP3, TP1/String	77705.51033	TCC (hras)/Full Cycle Time	136.83322	VRasc (AG-TP1) (m/min)	45.93624	2.17693
	110019.62353			VRasc (TR-HW) (m/min)	24.53928	48.90119
	66997.76093			VRasc (TR-TP1) (m/min)	24.53928	24.45060
Sarta3: TP3, TP2, TP1/String	94663.51033			VRasc (TR-TP2) (m/min)	24.53928	12.22530
	85556.64773			VRascen (m/min)	278.37564	91.49911
	66997.76093			Eficiencia FP para acarrear recortes(%)		78.02186

Volumetría Anular/Annular Volumetry		Optimal and Annular Speeds		Hydraulic Parameters	
V Anular Ag-DC(R)/Annular volume	3242.88000	Annular Velocity (AG-DC) (ft/min)	114.08926	Cleaning Index (Phi/pulg ²)	3.82019
V Anular Ag-HW(R)/Annular volume	2637.52000	Annular Velocity (AG-HW) (ft/min)	65.19386	Hydraulic Impact (lb)	874.25089
V Anular Ag-TP1(R)/Annular volume	3293.55000	Annular Velocity (AG-TP1) (ft/min)	56.16702	Equivalent Circulation Density (kg/cm ³)	1.57811
V Anular TR-HW(R)/Annular volume	63844.20000	Annular Velocity (TR-HW) (ft/min)	34.77006		
V Anular TR-TP1(R)/Annular volume	31922.10000	Annular Velocity (TR-TP1) (ft/min)	34.77006		
V Anular TR-TP2(R)/Annular volume	15961.05000	Annular Velocity (TR-TP2) (ft/min)	34.77006		
V Anular TR-TP3(R)/Annular volume	121101.3000	Optimal speed (ft/min) (Ag - Hta)	100.85470		
		Optimal speed (ft/min) (Tr - Hta)	82.51748		

Pressure System	
PTSist Circ (lb/pulg ²)	4556.22525
Pbarrena/Bit Pressure (kg/cm ²)	824.70840

Recuerde que el objetivo principal de la hidráulica de los fluidos de perforación es mantener el fondo del agujero libre de recortes y que estos se desplacen hacia la superficie de manera continua durante la perforación y circulación

Siguiete/Following

Fig. 6. Results of: string design, annular and optimal velocity, interior volume, circulation times and system pressure

These data are part of the design or user's guide used by Petróleos Mexicanos for each of its drilling projects, which is considered a standard in the presentation of projects for service companies. Likewise, this section is complemented by a

subsection that welcomes the user and briefly describes the content of the software and its procedures or hydraulic methods, in addition to indicating the technical literature used for its development.

Table 2. Results of hydraulic calculations using the national methodology

Result	Value
Drill pipe position evaluation	
Flotation factor (-)	0.8013
String weight (kg)	
Floated	83,391
1: Tp1-Tp2-Tp3	43,791
2: Tp2-Tp3-Tp1	77,706
3: Tp3-Tp2-Tp1	94,664
Determination of volumetries, times and speeds	
V_a (L)	121,101.3
V_i (L)	11,001.7
Pump expense (L/min)	1,850.5
Circulation Times (min)	
Time of delay	65.4
Full cycle time	136.8
Travel time	71.4
Annular velocities, v_a (ft/min)	
Ag-DC	114.1
Ag-HW	65.2
Ag-Tp1	56.2
TR-Tp1	34.8
Optimal velocities, v_o (ft/min)	
Ag-HTA	100.9
TR-HTA	82.5
Determination of pressure drops and cutting speed	
Nozzle area (in ²)	0.5890
ΔP_b (psi)	824.7 (58 kg/cm ²)
ΔP_a (psi)	4,441.8 (312 kg/cm ²)
ΔP_i (psi)	64.4 (4.5 kg/cm ²)
ν_r (m/min)	278.4 (4.6 m/s)
Fluid Carrying Efficiency (%)	78

2.2 Hydraulics I

Hydraulics I section is mainly developed by the methodology described by Petróleos Mexicanos (Pemex), [11, 12, 13]. Due to the experience in the field and in trainings by active personnel in the area of oil well drilling, all the above, is integrated as shown in the flow chart shown in Figure 1.

Hydraulics I calculations require entering casing and drillstring information such as initial and final lengths (L_i and L_f), inside and outside diameters (D_o and D_i), the grade and type of pipe in terms of its function (drill collar, heavy weight and drill pipe) and according to its strength (yield strength and tensile strength R).

In this regard, the typical grades, from lowest to highest strength, are E , X , G and S . This information allows generating the mechanical condition.

Once the mechanical state is appropriate, the program requests information on: i) The drilling fluid: density (ρ_f), yield point (σ_y), gel stress (G) and plastic, Marsh funnel and apparent viscosities (G) (μ_p , μ_e and μ_a), ii) Pump data: piston diameter (D_p), stroke length (L_p), volumetric efficiency (η), the pump operating cost (Q), iii) Cuttings data: length (L_r), diameter (D_r) and thickness (ϵ_r). With this new information the length of the tools is determined. The program comprises the following three cases depending on the pipe configuration:

$$a) L_{ag} = L_{DC} + L_{HW}$$

$$b) L_{ag} > L_{DC} + L_{HW}$$

$$c) L_{ag} < L_{DC} + L_{HW}$$

where L_{ag} , L_{DC} and L_{HW} are the lengths of the open hole, drill collar pipe and heavy weight pipe, respectively. The program then evaluates the drillstring design.

In general, the drillstring is composed of three types of pipes: drill collar, heavy weight and drill pipe. In particular, drill pipe has two subtypes: new pipe and (premium) used pipe. The calculation procedure determines the design of the drillstring and casing.

From this design, the number of sections, the annular volume (V_a), the interior volume of the drillstring (V_i), the internal pressure drops of the string and in the annular volume (ΔP_i and ΔP_a and the fluid velocities in the annular space and optimal (v_a and v_o).

Furthermore, the adjusted weight (W_{adj}), bit pressure drop (ΔP_b) and nozzle variables, such as nozzle diameter (D_t), fluid velocity at the nozzle outlet (v_t), hydraulic impact and hydraulic factor

Método 1/Method 1 | Volumetry and Hydraulics | Método 2/Method 2 | Volumetry and Hydraulics | Rheological Models

Informe de Tuberías de Revestimiento/Casing information

T Revestimiento Casing	Diámetro Ext (") External diameter	Diámetro Int (") Inside diameter	Prof. Inicial (ft) Initial Depth	Prof. Final (ft) Final Depth
Tr Cond-Hinc Driver	0	0	0	0
Tr Superficial Surface	0	0	0	0
Tr Intermedia Intermediate	13.375	12.5	0	2135
Tr Explotación Exploitation	9.625	8.835	0	10786
Ag descubierto Bare hole	0	8.625	10786	12031

Información Básica/Basic information

Profundidad (ft)/Depth	12031
Diámetro Bna (")/Bit Diameter	8.625
Toberas (1/32")/Nozzles	11 11 11
Presión Superficial (psi)/Surface pressure	3000
Peso del lodo (lb/gal)/Mud Density	12.8
Viscosidad Embudo (seg/qt)/Funnel Viscosity	42
Viscosidad Plástica (cP)/Plastic Viscosity	19
Punto Cedente (Lb/100ft ²)/Yield Point	15
Esfuerzo Gel (Lb/100ft ²)/Gel Stress	8
Caudal (gpm)/Flow	335

Datos de Sarta de Perforación/Drill String Data

Hta Tool	Diámetro Ext (") External Diameter	Diámetro Int (") Inside Diameter	Prof. Inicial (ft) Initial Depth	Prof. Final (ft) Final Depth	Grado (°) Grade	P Ajustado (lb/ft) Adjusted Weight
DC	7	2.25	12031	11641		
HW	0	0	0	0		
TP1	4.5	3.826	11641	0		
TP2	0	0	0	0		
TP3	0	0	0	0		

Conexiones Superficiales/Surface Connections

Caso Case	Tubo Vertical (ft, ") Vertical Tube	Manguera (ft, ") Hosepipe	Unión Giratoria (ft, ") Swivel Union	Kelly (ft, ") Kelly	Long. Equipo (ft, ") Equipment Length
3	45	4	55	3	25
				2.5	40
				3.25	610
					3.826

Estado Mecánico Hidráulica/Hydraulic Wellbore Program

Fig. 7. Hydraulics II data entry section

(IH and FH), cuttings upward velocity (v_r), among other variables and parameters, are estimated.

2.3 Hydraulics II

Similarly, the Hydraulics II section is based on a combination of the methodologies described in [4]. Field experience and training material from international companies, which can be found in the flow chart in Figure 2.

2.4 Rheological Models

Finally, the section Rheological Models is integrated by the results of the Fann viscometer test with which rheology values are calculated for low and high shear rate using the Bingham plastic and exponential law models [6].

Drilling fluid hydraulics refers to the fluid that performs a path that starts its displacement by means of mud pumps, which can be duplex or triplex, which are considered the heart of the circulatory system and that concludes in the settlement dam.

For the realization of the program, it was necessary to know the diagram or fluid circulation

circuit during the drilling or circulation stages, as shown in Figure 4. The flow diagram of the calculation processes allows to identify the sections and the sequence of the programming, which have the following order:

1. Schematize the diameters and depths of the casing pipes or casing.
2. Draw the pipes with diameters and depths that integrate the drill string. Establish the minimum necessary information such as: mud pump characteristics, drilling fluid type and density, fluid rheology, drill cuttings characteristics and types of connections.
3. Corroborate that the annular spaces and string interior correspond to the mechanical condition (MS) design.
4. Relate input data to annular volumetry and string interior calculations.
5. Identify the possible cases according to the MS design.
6. Develop the hydraulic method according to the input data.

Table 3. Information required by the program (API methodology)

Element	Dimensions and properties				
Casing Pipe	D_o (in)	D_i (in)	Depth L (ft)		
Intermediate	13.375	12.500	2,135		
Exploitation	9.625	8.835	10,786		
Drill string					
Hole	8.625		10,786 – 12,031		
Drill collar	7.000	2.250	12,031 – 11,641		
TP1	4.500	3.826	11,641		
Surface connection (case 3)	D_o (in)	Length (ft)			
Vertical Pipe	4.00	45			
Hose	3.00	55			
Swivel joint	2.50	25			
Kelly device	3.25	40			
Equipment length	3.82	610			
Nozzles	N_t	D_t (in)			
	3	11/32			
Fluid properties	ρ (lb/gal)	σ_y (lb/ft ²)	G (lb/ft ²)	μ_a (cP)	μ_e (s/qt)
	12.8	0.15	0.08	19	42
Other dimensions					
Total depth (ft)	12,031				
Surface pressure (psi)	3,000				
Bit diameter (in)	8.625				

Método 1/Method 1 | Volumetry and Hydraulics | Método 2/Method 2 | Volumetry and Hydraulics | Rheological Models

Basic Operations				Complementary Hydraulic Calculations				
Rheology Data	String and Annular	Parameter	Surface Connections					
L600 53.00000	ns	0.64009	Propagation Velocity (ft/min)	560.22991	Equivalent Circulation Density (lb/gal)	13.12849	Impact Force/pulg ² (PSI)	14.60379
L300 34.00000	Ks	3.20906	Effective Viscosity (cP)	48.98212	Pbna (PSI)/Pressure Bit	1700.63429	Hydraulic Power on Bit (hhp)	327.42097
L100 21.33333	na	0.26653	Reynolds Number	8663.42727	%Pbna/% of Pressure in the Bit	56.68781	P H/pulg ² (hhp/pulg ²)	5.58975
L3 8.00000	Ka	26.46644	Friction Factor	0.00603	Vel.Tobera (ft/seg)/Nozzle Speed	385.01928	Total System Pressure (PSI)	3018.34133
			Pressure Drop (PSI)	41.54156	Impacto H (lb)/Hydraulic Impact	855.42108		
				Calcular / Calculate				

Cálculos en Sarta de Perforación/Drill String Calculations							
Propagation Velocity (ft/min)	1619.91111	Propagation Velocity (ft/min)	0	Propagation Velocity (ft/min)	560.22991	Propagation Velocity (ft/min)	0
Effective Viscosity (cP)	27.61147	Effective Viscosity (cP)		Effective Viscosity (cP)	48.98212	Effective Viscosity (cP)	
Reynolds Number	26133.65907	Reynolds Number		Reynolds Number	8663.42727	Reynolds Number	
Friction Factor	0.00444	Friction Factor		Factor Fricción	0.00603	Factor Fricción	
Pressure Drop DC (PSI)	277.89403	Pressure Drop HW (PSI)		Pressure Drop TP1 (PSI)	792.76278	Pressure Drop TP2 (PSI)	
				Calcular 2/ Calculate			
				Total Pressure (PSI) 1070.65681			

Cálculos Agujero-Herramienta/Hole-Tool Calculations							
Uncovered Hole Length = LDC+LHW				Uncovered Hole Length>LDC+LHW			
Propagation Velocity (ft/min)	322.98535	Propagation Velocity (ft/min)	0	Propagation Velocity (ft/min)	0	Propagation Velocity (ft/min)	141.86462
Effective Viscosity (cP)	34.15139	Effective Viscosity (cP)		Effective Viscosity (cP)		Effective Viscosity (cP)	128.24560
Reynolds Number	3042.59288	Reynolds Number		Reynolds Number		Reynolds Number	949.37276
Factor Fricción	0.00468	Factor Fricción		Factor Fricción		Factor Fricción	0.02528
Pressure Drop AG-DC (PSI)	16.14176	Pressure Drop Ag-TP (PSI)		Pressure Drop AG-HW (PSI)		Pressure Drop TR-TP (PSI)	174.38674
				Calcular 3/Calculate			
				Total Annular Pressure (PSI) 205.50867			
				Siguiente/Following			

Fig. 8. Determination of coefficients

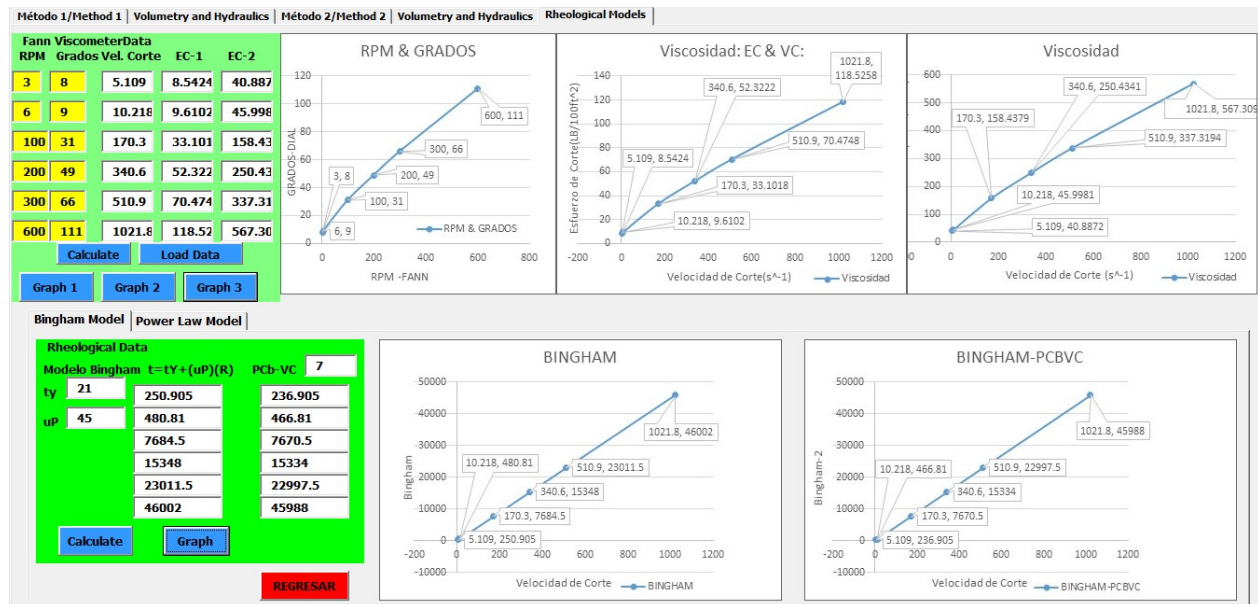


Fig. 9. Shear rate and shear stress calculations

7. Apply the Fann viscometer test values in the rheological models.

In the applied hydraulics calculation procedure, the flow rate and string consistency parameters (n_s and k_s) are estimated, where the former describes the degree of shear thinning of a fluid and is dimensionless, depends on the viscosifier quality and is controlled with chemical thinners.

On the other hand, the consistency index is considered as plastic viscosity, either by an increase in the concentration of solids or a decrease in the particle size.

Its control is achieved with mechanical solids control and dilution equipment. Additionally, these indices are also calculated for the annular space, denoted as n_a and k_a respectively.

Additionally, the calculations of average propagation velocity, effective viscosity, Reynolds number, Fanning friction factor and pressure loss are determined for each of the drillstring and annular space interior volumes generated from the mechanical state design.

The friction factor is determined by considering the value of the Reynolds number (Re).

Depending on whether the flow regime is predominantly laminar or turbulent, the friction factor (F_f) is calculated by: For the case of predominantly laminar flow ($Re < 2100$):

$$F_f = \begin{cases} \frac{24}{N_{Re}} & Re \leq 2100 \\ \frac{\log n + 3.93}{\frac{50}{\left(\frac{1.75 - \log n}{7}\right)}} & Re > 2100, \end{cases} \quad (1)$$

where n was previously determined for the inner string volume and annular space.

The calculations in the Complementary Hydraulic Calculations section correspond to drilling optimization, i.e., estimating a penetration rate according to bit performance [4], since among the many factors that can modify it are, for example:

Bit size, type and characteristics, type and concentration of solids in the formation, and bit hydraulics. However, the latter can be optimized by controlling hydraulic power and nozzle velocity.

Table 4. Hydraulic calculation results using API methodology

Result	Value		
Rheological parameters	String	Annular space	
Flow index n	0.64	0.26	
Consistency index k	3.20	26.46	
Fann Viscometer readings			
@ 600 rpm			53
@ 300 rpm			34
@ 100 rpm			23.1
@ 3 rpm			8
Variables of surface connections			
Propagation speed (ft/min)			560.23
Effective viscosity (cP)			48.98
Re			8,663
F_f			0.0060
Surface connection pressure (psi)			41.54
Drillstring variables	Drill collar	Tp-1	
Propagation speed (ft/min)	1619.91	560.23	
Effective viscosity (cP)	27.61	48.98	
Re	26,133	8,663	
F_f	0.0044	0.0060	
Pressure (psi)	277.9	792.8	
Variables for annular sections	Ag-DC	Ag-TP	TR-TP
Propagation speed (ft/min)	322.98	151.47	141.86
Effective viscosity (cP)	34.15	117.86	128.24
Re	3,042	1,049.5	949
F_f	0.0047	0.0229	0.0253
Pressure (psi)	16.1	15.0	174.4

3 Results

Application Examples

To demonstrate the capabilities of the software, two illustrative examples of oil well drilling have been selected. First, a calculation procedure using the national methodology and, subsequently, an exercise developed under the API methodology.

Introduction Section and Description of Hydraulics

Figure 3 corresponds to the front page of the software, which requests basic information from the design guide, in order to know the name, number, location and objective of the well, as well as the name of the technician or design engineer.

To perform the first hydraulic calculations, it is required to enter a set of data whose information is classified into seven elements: casing, borehole, drillstring, pump, mud properties, nozzles and cuttings.

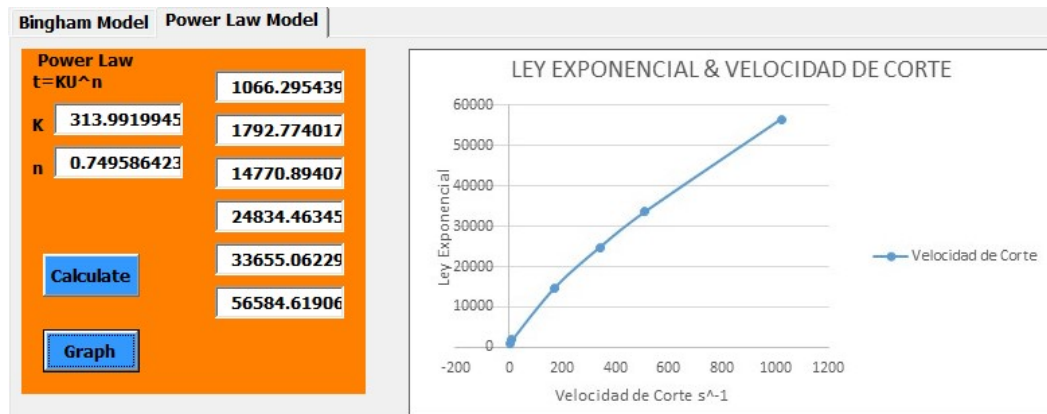
Table 1 provides the necessary information to exemplify the use of this tool using national petroleum engineering methods and procedures.

Technical Data and Mechanical State Section

Figure 5 corresponds to the input of technical data of the casing pipes, drillstring, mud pumps, bits, solids/cuttings and the drilling fluid whose first result corresponds to the generation of the mechanical state, in which it can be identified the number of annular spaces both in the hole section and in the casing pipes.

Table 5. Results of complementary hydraulic calculations

Result	Value
DEC (lb/gal)	13.13
P_b (psi)	1,700
% P_b (%)	56.7
v_t (ft/s)	385
IH (lb)	855.4
IH/in^2 (psi)	14.6
Hydraulic bit power (hhp)	327.4
Hydraulic bit power $/in^2$ (hhp/ in^2)	5.6
Total system pressure (psi)	3,018

**Fig. 10.** Exponential law rheological model

The drillstring is outlined with a color code: the blue color is the drill collar, the gray color is the heavy weight, and the orange, green and brown colors correspond to the drill pipes in the order Tp1, Tp2 and Tp3, respectively.

It is important that the design engineer or technician corroborate the MS schematic in this section as it determines the volume of drilling fluids inside the string and the corresponding hole sections, casing pipes, and drillstring tools.

Hydraulics I Section

Figure 6 begins with the evaluation of the drill pipes (Tp), the pull margin, the flotation factor, and the weight of the floating drillstring.

Subsequently, the calculations according to the MS design are obtained, such as: hole-string annular volumetry, internal volume of each of the tools that make up the drillstring, expenses and circulation times, annular and optimal speeds, speed and area of nozzles, internal pressure in the drillstring, hydraulic parameters, pressure in annular intervals, transport of cuttings, among others.

Each of the previous points has its respective physical units and four decimal places of precision, as shown in Table 2.

Accurate estimation of pressure drop is important for the design of the drillstring, nozzles and for optimizing fluid circulations, as well as identifying drilling problems such as flushing or clogging of the bit nozzles [2].

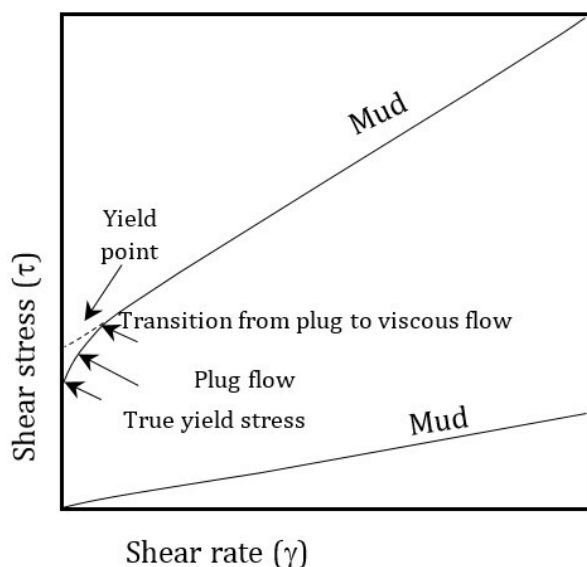


Fig. 11. Flow diagram of Newtonian and typical mud. Adapted from API Energy Handbook (2001)

Hydraulics II Section

To illustrate the use of the Hydraulics II Section, referring to the hydraulic methodology established in [6], the data will be entered as shown in Figure 7.

This methodology is used by transnational oil companies, the information indicated in Table 3 is required. The exercise used corresponds to the example of the API manual.

For this procedure it is required to know the results of the Fann viscometer tests and the value of the total pressure of the surface system, and to identify the corresponding case of the types of surface connections with which the drilling equipment is operating.

Figure 7 right, schematizes the mechanical state according to the data of the various casing pipes and tools that make up the drillstring. Figure 8 shows the solution of the exercise.

Figure 8 corresponds to the results of the applied hydraulics operations, where initially the readings are determined at different rotational speeds of the viscometer, i.e. 600, 300, 100, and 3 rpm, where the lowest speed corresponds to a low shear stress.

From these data, the flow and consistency indices of both the string and the annular space (n_s , k_s , n_a y k_a) are calculated.

Once the type of surface connection is identified, it is essential to know that it is integrated by the vertical pipe, the kelly device hose, swivel joint and the kelly "traveling rotary"; but as there is no standard value for the traveling rotary it is important to know the geometry of each of them, for example, the vertical pipe and hose measure approximately 86 ft in length, with an inside diameter of 3 or 3.8 in and the S-hose is different for each rig, giving a total of four connection cases, as reported in the API manual [6]. For this exercise, Table 3 uses case 3 as stated in the manual.

The resulting hydraulic calculations are summarized in Table 4. After the mechanical state is generated, the Fann viscometer readings are entered to determine the n and k parameters for the string and for the annular space.

Finally, to complete the hydraulic design of the well, and regardless of the number of sections and the type of pipe to be used, it is essential to perform complementary calculations, which are summarized in Table 5.

Rheological Models Section

Figure 9 corresponds to the Rheological Models Section, which allow describing the relationship between shear stress and shear velocity. Most drilling fluids are non-Newtonian fluids, which is why Newton's viscosity law or model does not describe their flow behavior. For this program, the Bingham plastic and exponential law models were implemented.

In recent years, the Bingham plastic flow model has been one of the most widely used to describe the flow characteristics of drilling fluids; however, this model is characterized by requiring a finite force to initiate the flow to subsequently develop a constant viscosity as the shear rate increases.

The curve or profile of a typical or conventional fluid is obtained from the rotary viscometer data, where this curve does not pass through the origin point, as shown in the graphs in Figure 9 and Figure 11, between shear rate and shear stress.

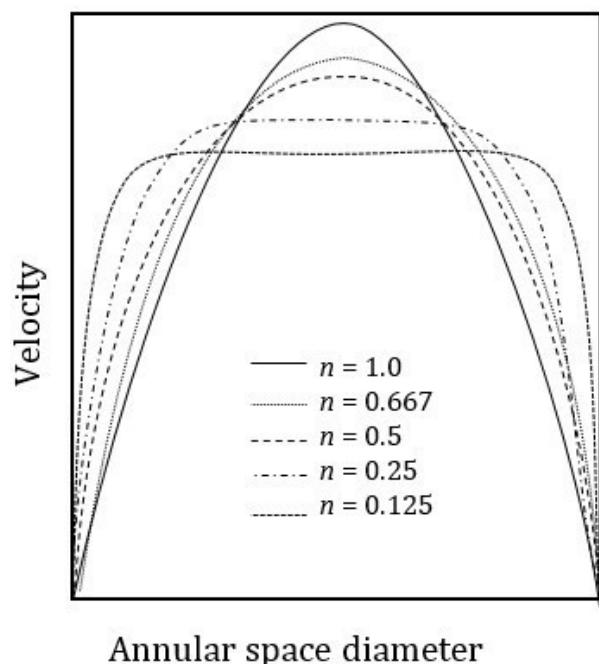


Fig. 12. Effect of Exponential Law index n on velocity profile. Adapted from API Energy Handbook (2001)

In Figure 10, the exponential law model does not assume that there is a linear relationship between shear stress and shear rate; however, for fluids obeying the exponential law, its curve starts from the origin.

In this model it is confirmed that the shear rate varies according to the change in shear stress. The Bingham rheological and exponential law models are among the most used and important for the determination of pressure loss in the drilling system [16].

Finally, for the Hydraulics II section, the value of n_a defines the flow profile formed in the annular space (Figure 12); i.e., n_a must have a value close to zero for its profile to be flat, which is considered ideal for the transport of cuttings towards the surface.

Nevertheless, if the value of n_a is very close to 1, the velocity profile will be very steep in the center, resulting in cuttings being displaced towards the annular space walls, thus increasing the possibility of drill string entrapment.

In addition, this methodology allows to contrast the result with the initial surface pressure value, allowing to know more quickly if the applied hydraulics is in accordance with what the surface monitoring equipment indicates.

4 Conclusions

The computer model developed in the Microsoft Visual Basic environment is an efficient and useful tool for the calculation of various hydraulic aspects related to oil well drilling. The interface design is simple and intuitive, of fast communication and easy adaptation between program and user.

The computer program integrates two hydraulic methodologies in a didactic way, but keeping in each of them its originality, development philosophy and in accordance with the field operation. In the program, the existing theory in the specialized literature is combined.

The mechanical state schematization considers the scaled relationships of casing and drillstring dimensions, and knowledge of the mechanical state is essential for consideration, analysis and interpretation of hydraulic methods.

In the future, this model will be enhanced to allow the user to make custom modifications based on their experience or internal company design criteria; for example, propose bit, annular space or drillstring pressure drops, or modify nozzle diameters and make adjustments based on the maximum hydraulic power or maximum hydraulic impact method.

Additionally, this computer program considers the two most commonly used rheological models, although in a later version new models will be included so that the user has greater flexibility according to the information available in his design project.

This tool aspires to become the most widely used software in higher education institutions where new technicians and engineers are trained, as well as a support tool in training courses for the oil well drilling industry.

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